

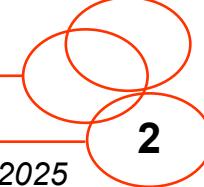
« EMR AND INVERSION-BASED CONTROL OF AN ELECTRIC VEHICLE »

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- Outline -



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Studied EV

2

EMR of the studied EV

3

Inversion-based control of the EV

1. Studied EV

- Simplified description of a Renault Zoe -

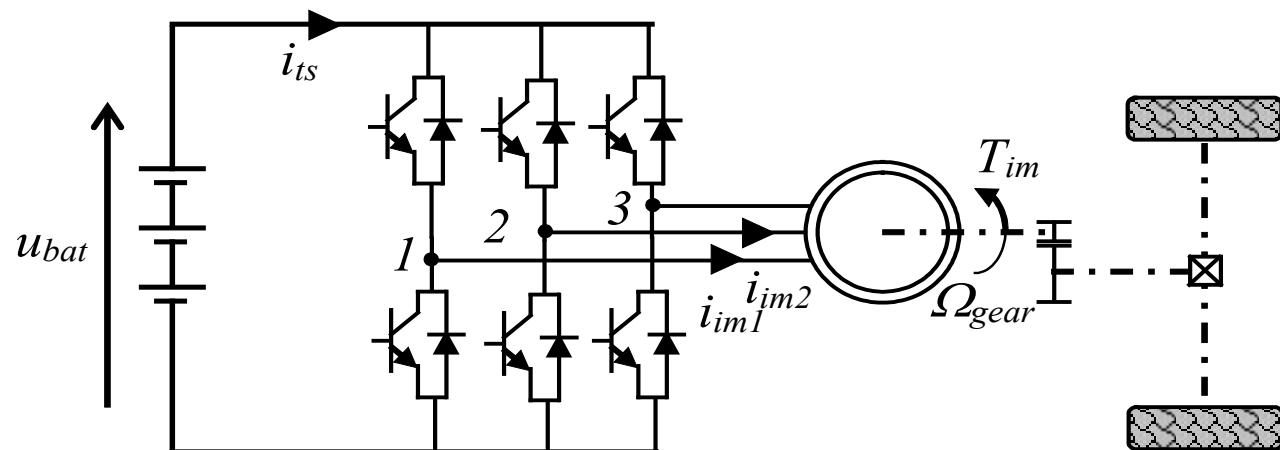
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Renault Zoe main characteristics (2018)

- 80kW – 100kW Synchronous Motor
- 41Ah Li NMC battery
- 1555 kg (empty mass)



Objective:

- Illustration of EMR-IBC methodology: control of the traction system in straight lines
- Introduction to simulation session

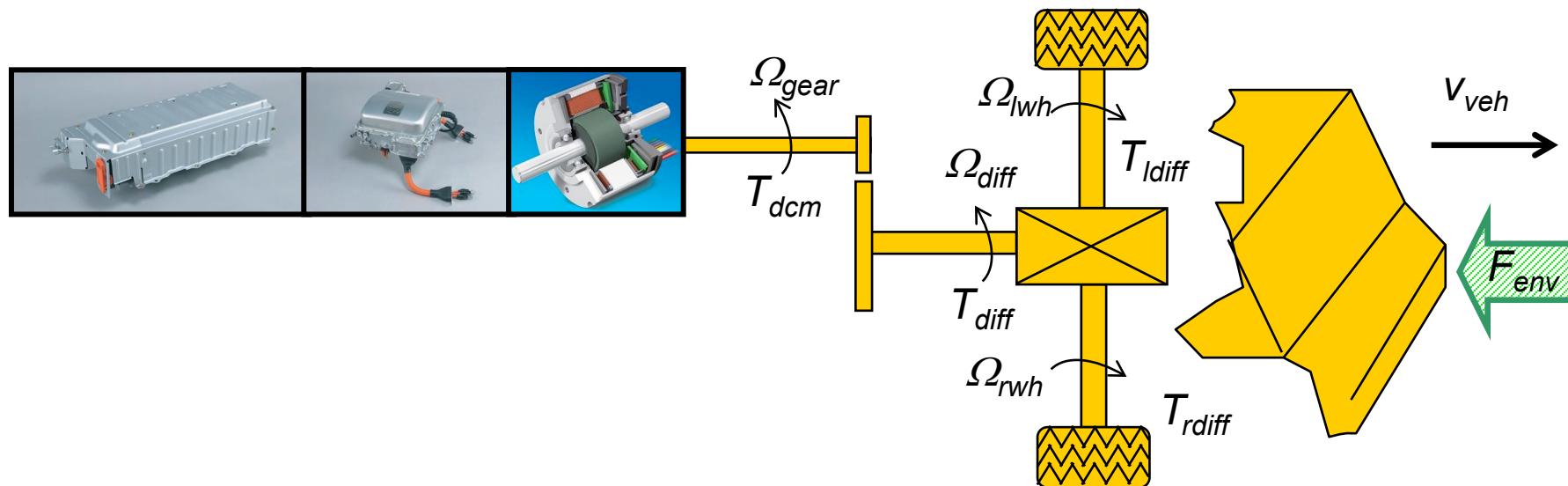
Simplifications and assumptions for analysis and EMR

- A non-saturated permanent magnet DC machine is considered (instead of Synchronous machine)
- Ideal power converter (no losses)
- Inertia of rotating masses are not taken in account
- Contact wheel/ground with no slip and pneumatic deformation
- Mechanical brakes are not considered

- Structural representation -

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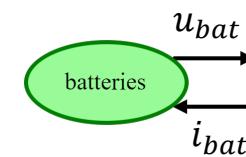
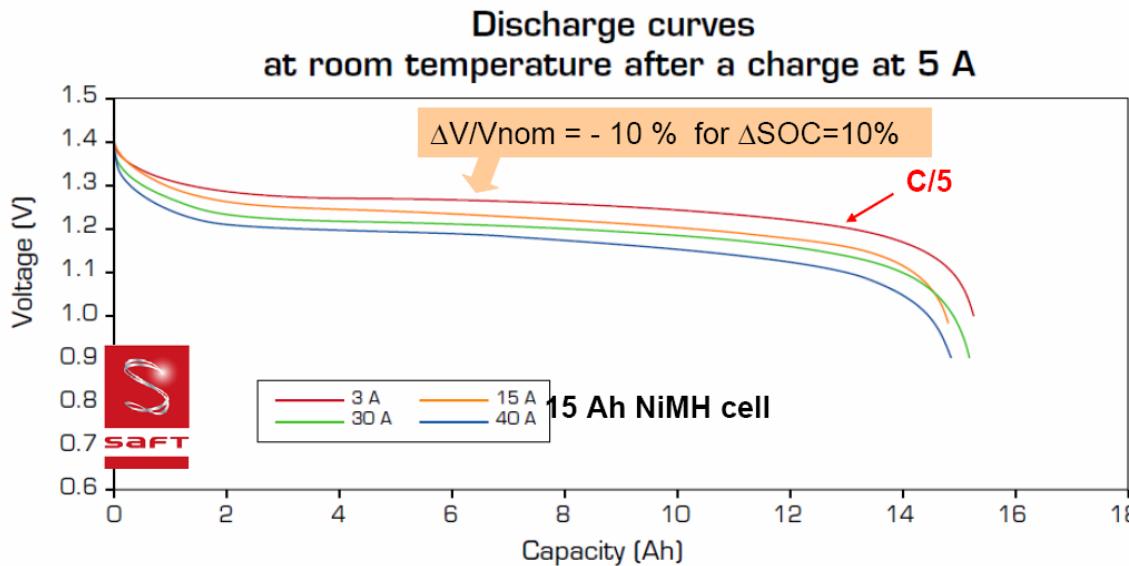
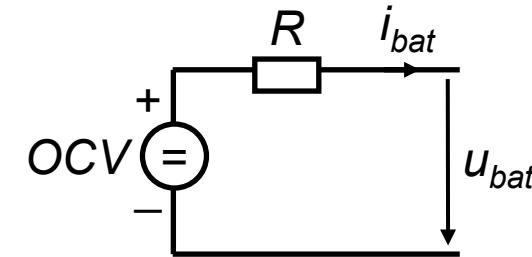


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EMR of the studied EV

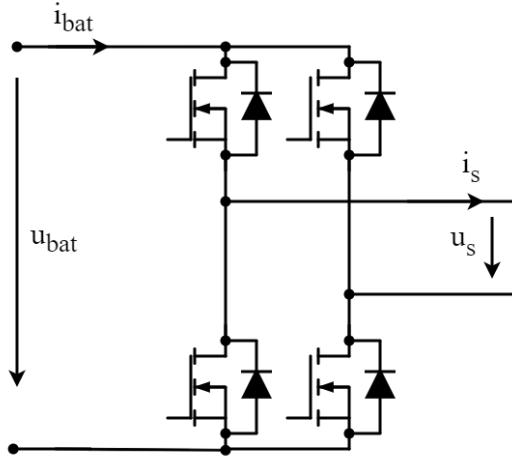
- Batteries -

- Terminal element of system – Environnement of system
 - Must be modelled anyway
 - Open Circuit Voltage (OCV)
 - Internal impedance (R):
 - State of Charge (SOC)



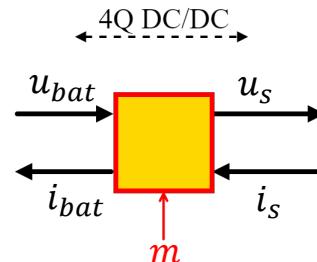
- Power Converter and DC machine -

- 4Q DC/DC converter for full reversibility
 - Introducing d as the main duty cycle
 - Definition of an average model ($0 < d < 1$)

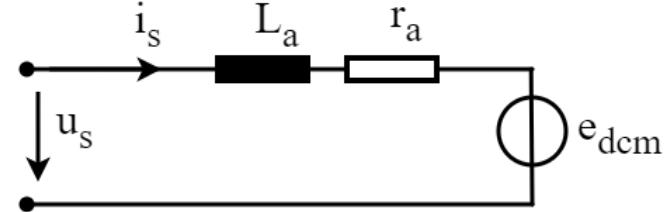


$$\begin{cases} u_s = \underbrace{(2 \cdot d - 1)}_m \cdot u_{bat} \\ i_{bat} = \underbrace{(2 \cdot d - 1)}_m \cdot i_s \end{cases}$$

with $-1 < m < 1$

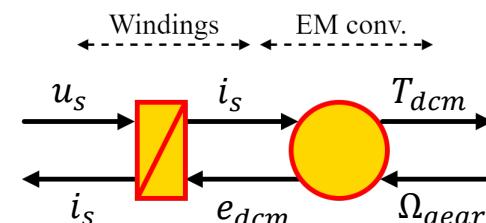


- DC machine with permanent magnets
 - Electrical equations



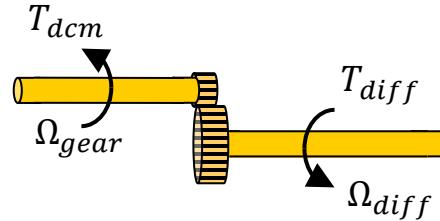
- Electro-mechanical conversion

$$\begin{cases} T_{dcm} = k_{dcm} \cdot i_s \\ e_{dcm} = k_{dcm} \cdot \Omega_{gear} \end{cases}$$



- Gearbox and differential -

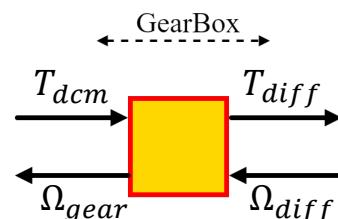
- Gearbox



- k_{gear} : transformation ratio
- η_{gear} : efficiency, p : correction exponent

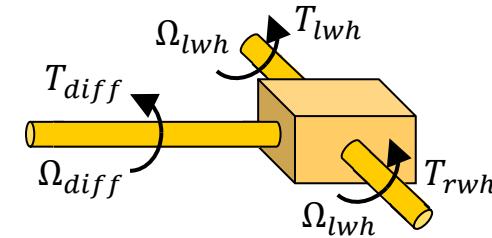
$$\begin{cases} T_{diff} = \eta_{gear}^p k_{gear} T_{dcm} \\ \Omega_{gear} = k_{gear} \Omega_{diff} \end{cases}$$

with $\begin{cases} p = 1 & \text{if } \Omega_{diff} T_{diff} \geq 0 \\ p = -1 & \text{if } \Omega_{diff} T_{diff} < 0 \end{cases}$



- Differential

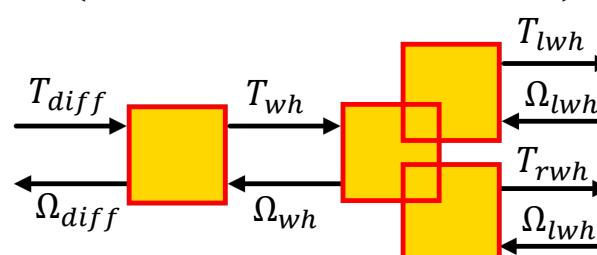
- 2 functions in 1



- k_{diff} : transformation ratio
- η_{diff} : efficiency, p : correction exponent

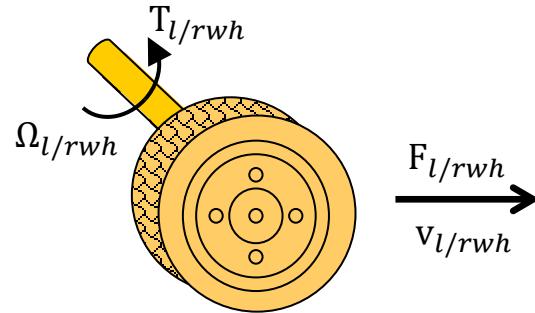
$$\begin{cases} T_{wh} = \eta_{diff}^p k_{diff} T_{diff} \\ \Omega_{diff} = k_{diff} \Omega_{wh} \end{cases} \text{ and } \begin{cases} T_{ldiff} = T_{rdiff} = \frac{1}{2} T_{wh} \\ \Omega_{wh} = \frac{1}{2} (\Omega_{lwh} + \Omega_{rwh}) \end{cases}$$

Differential



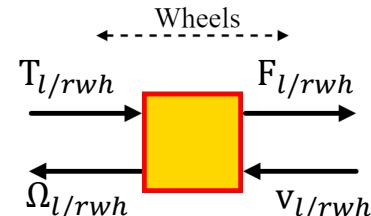
- Wheels and ground coupling -

- Wheels



– R_{wh} : wheels radius

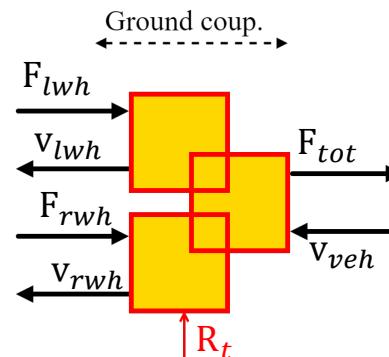
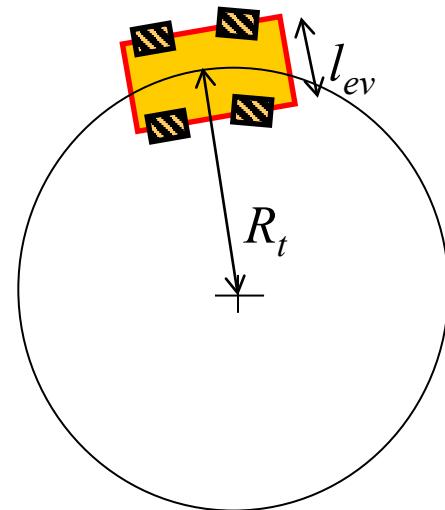
$$\begin{cases} F_{l/rwh} = k_{wh} \cdot T_{l/rwh} \\ \Omega_{l/rwh} = k_{whf} \cdot v_{l/rwh} \end{cases} \quad \text{with} \quad k_{wh} = \frac{1}{R_{wh}}$$



- Ground coupling

- R_t : turning radius
- L_{ev} : vehicle width

$$\begin{cases} v_{lwh} = \left(1 + \frac{1}{2} \cdot \frac{l_{ev}}{R_t}\right) \cdot v_{veh} \\ v_{rwh} = \left(1 - \frac{1}{2} \cdot \frac{l_{ev}}{R_t}\right) \cdot v_{veh} \\ F_{tot} = F_{lwh} + F_{rwh} \end{cases}$$



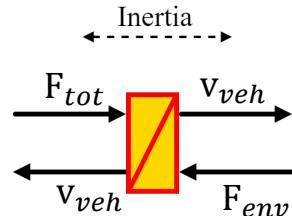
- Main Inertia and Environment -

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Inertia (Chassis)

- M_{veh} : total mass of the vehicle

$$M_{veh} \cdot \frac{d v_{veh}}{dt} = F_{tot} - F_{env}$$

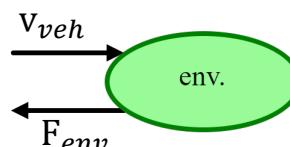
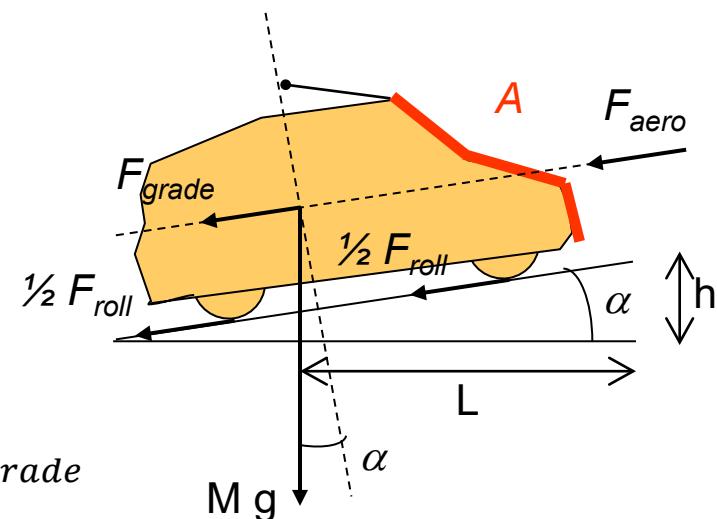


- Environment - Terminal element of system – Must be modelled anyway

- F_{aero} : aerodynamic resistance
- F_{roll} : rolling resistance
- F_{grade} : grade resistance

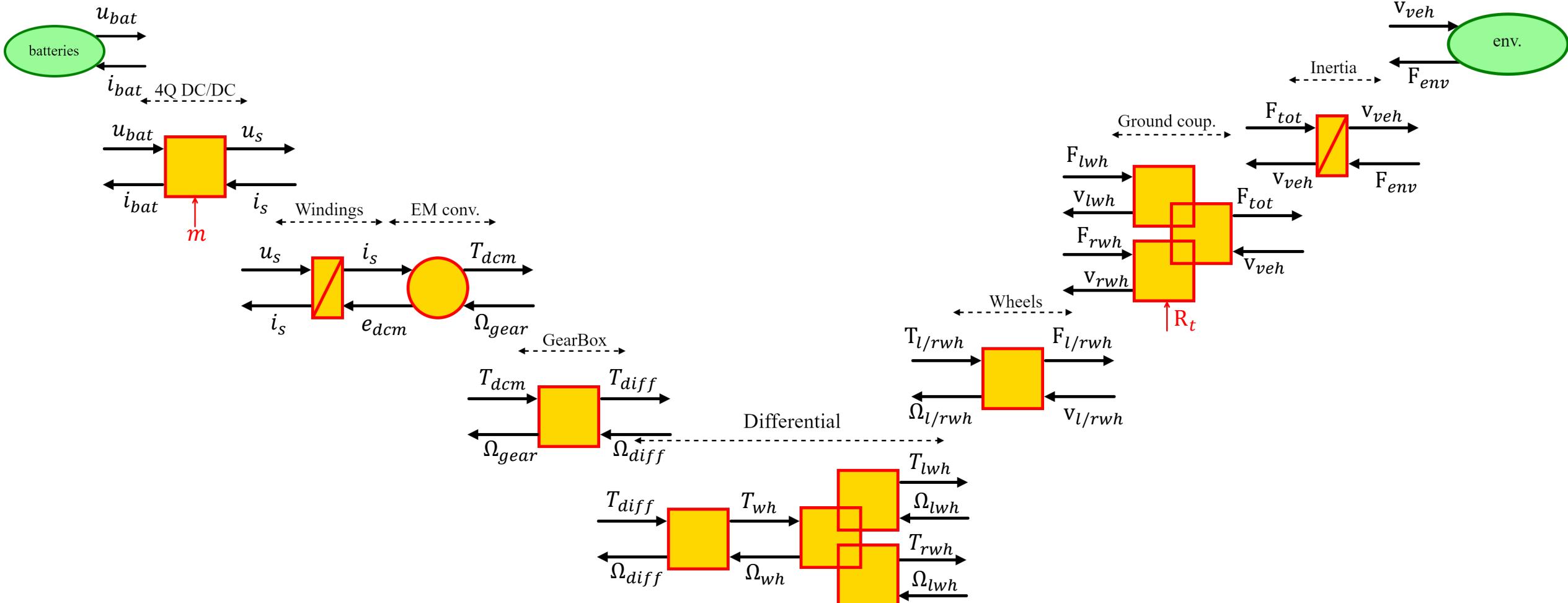
$$\begin{cases} F_{aero} = \frac{1}{2} \rho_{air} \cdot A \cdot C_x \cdot v_{veh}^2 \\ F_{roll} = k_{roll} \cdot M_{veh} \cdot g \cdot \cos\alpha \\ F_{grade} = M_{veh} \cdot g \cdot \sin\alpha \end{cases}$$

$$F_{env} = F_{aero} + F_{roll} + F_{grade}$$



- From subsystems analysis to systemic representation -

- All subsystems have been modelled and represented
 - They can be associated as far as: IOs for one element are compatible with IOs of the consecutive elements

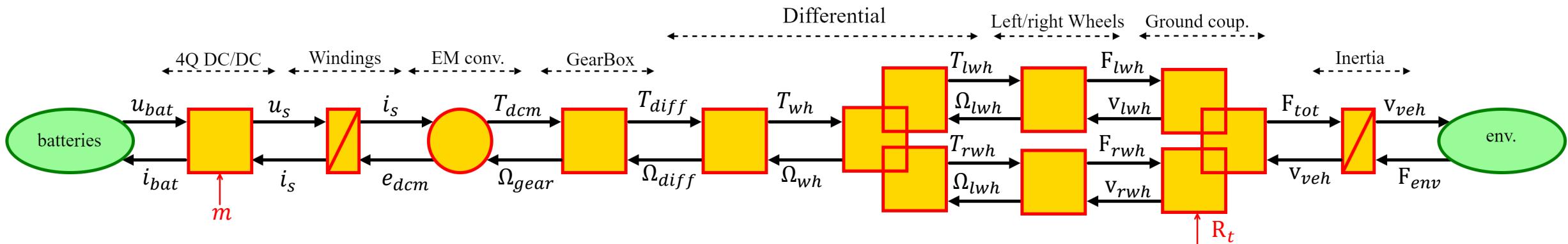
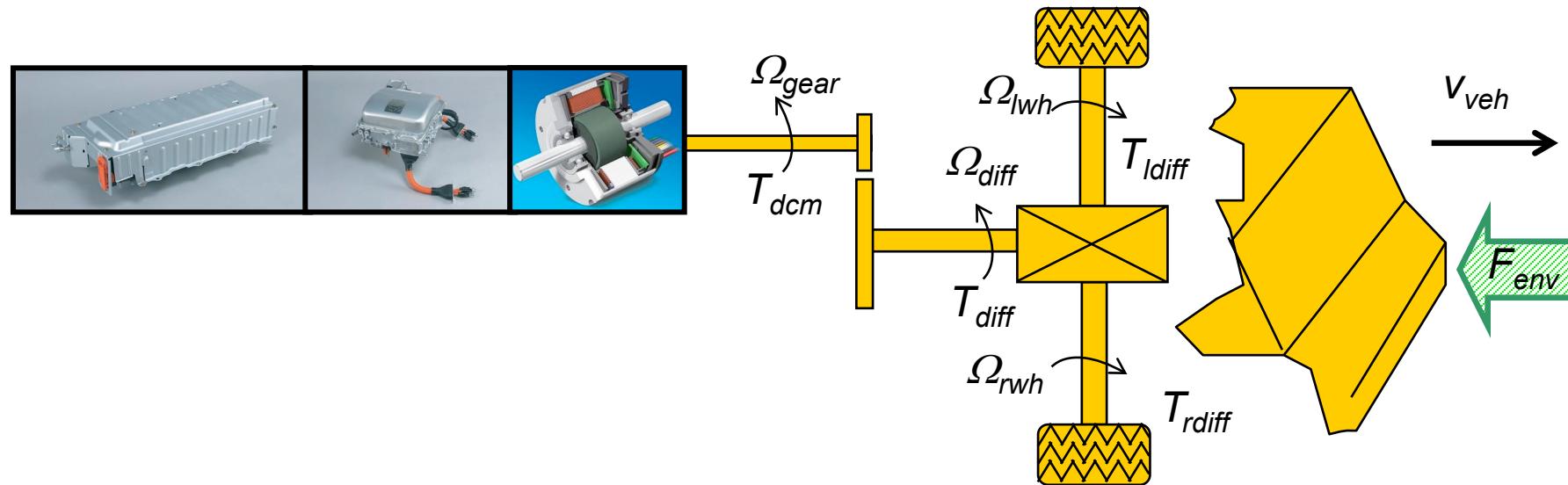
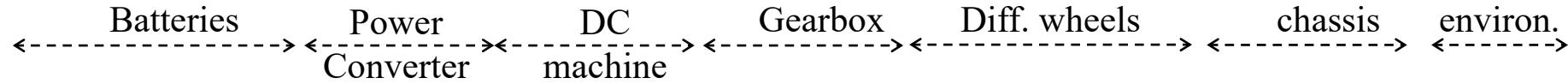


EMR and control of an EV

- Structural vs. Functional representation -

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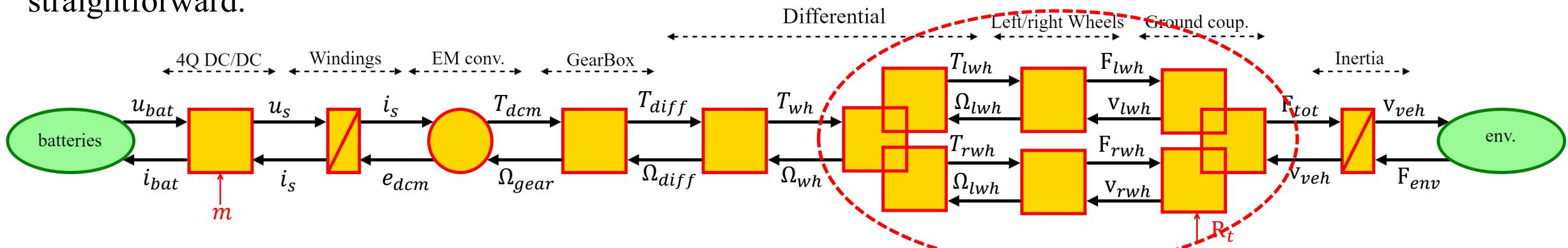
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- Final EMR -

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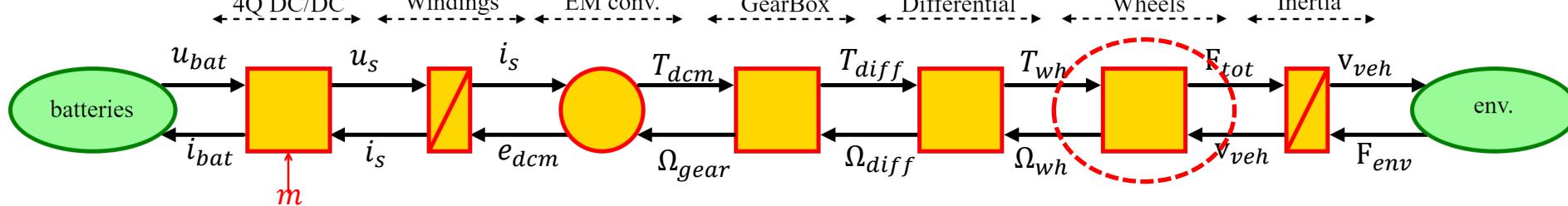
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- Simplifications under conditions: Contact wheel/ground with no slip and pneumatic deformation, the vehicle goes straightforward.



If the vehicle drives in a straight line ($R_t = \infty$), an equivalent wheel is sufficient

model reduction

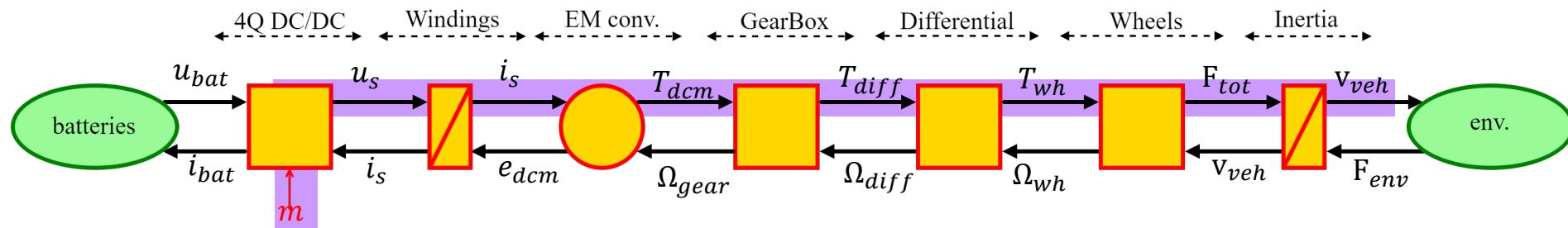


$$\begin{cases} F_{tot} = \frac{1}{R_{wh}} T_{wh} \\ \Omega_{wh} = \frac{1}{R_{wh}} v_{eh} \end{cases}$$

Inversion-based control of the studied EV

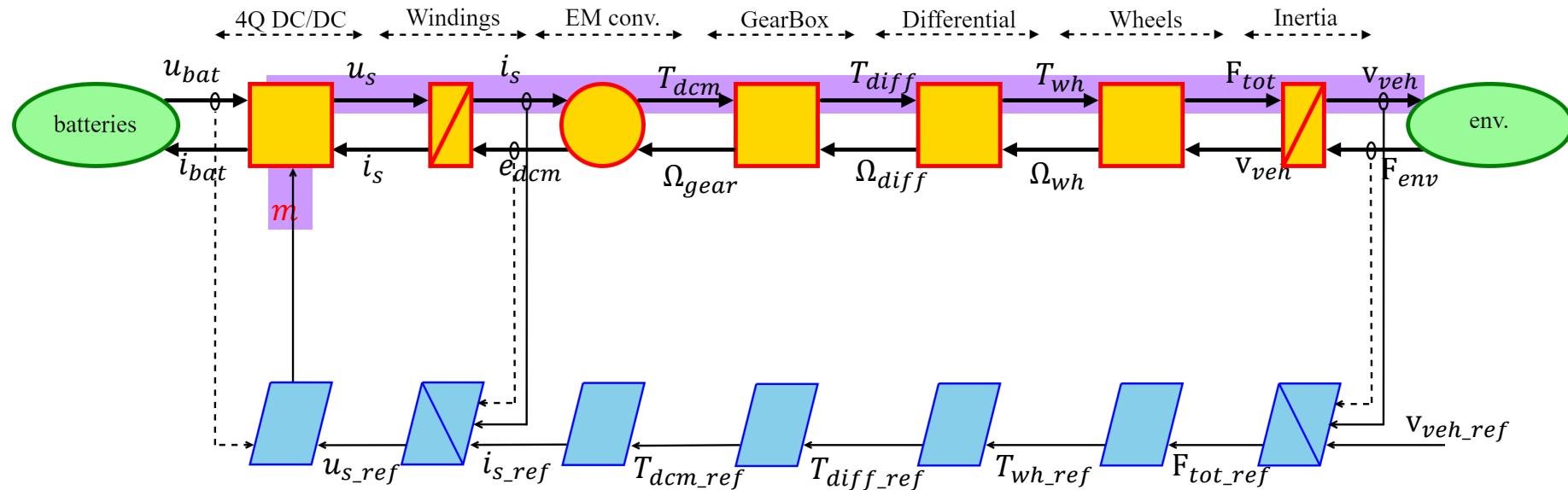
- Tuning path Identification -

- Objectives: control of vehicle speed
- Constraints: DC machine current limitation (Torque limitation)
- Tuning variable: modulation function of the 4Q converter
 - Tuning path: from the tuning variable to the objective through constraints



- Inverse Based control -

- Inversion of the EMR's éléments, from the objectif to the tuning parameter

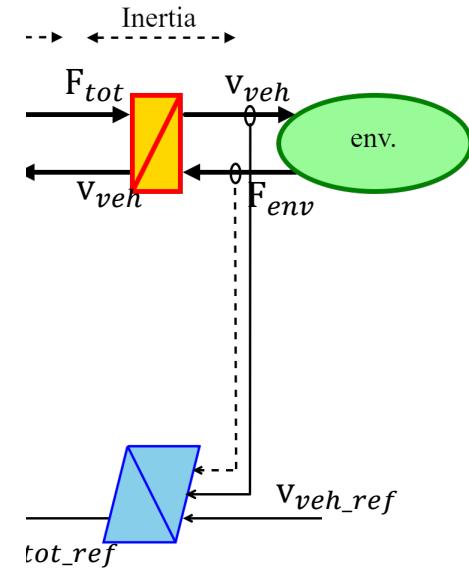
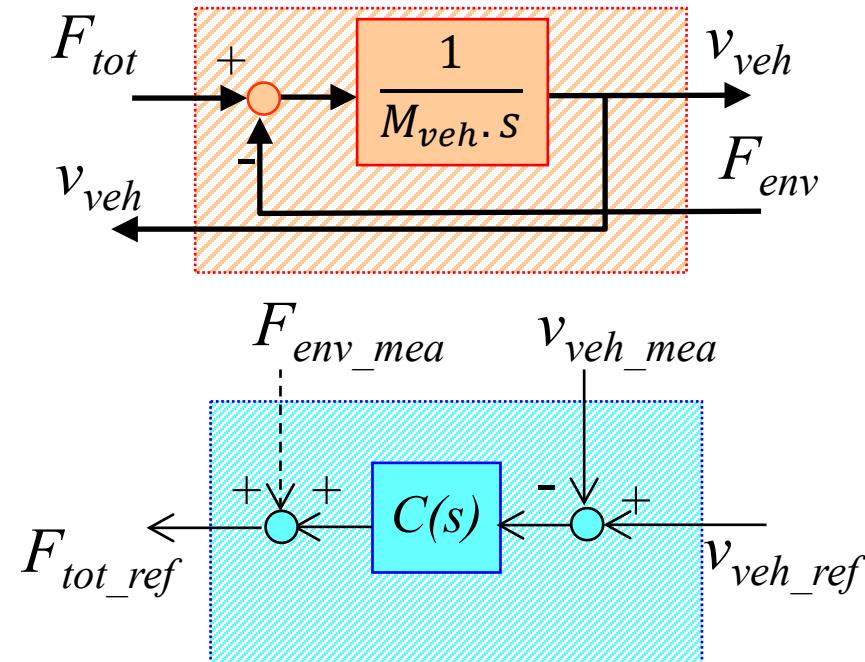


Maximum Control Structure:

- inversion of each element step-by-step
- all variables are supposed to be measurable

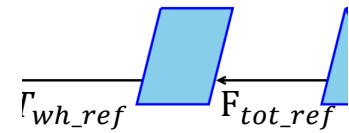
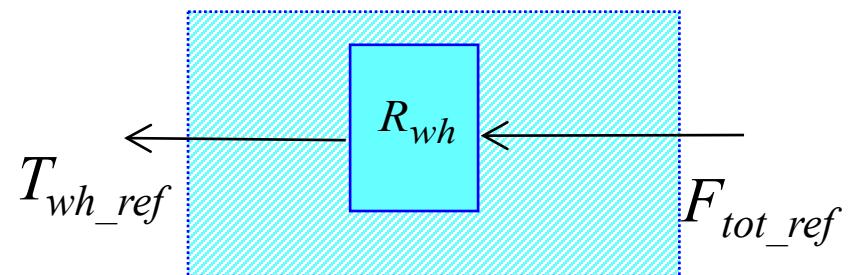
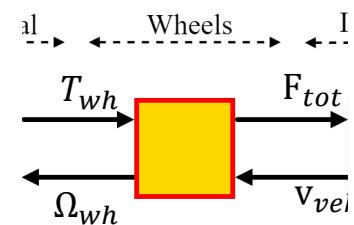
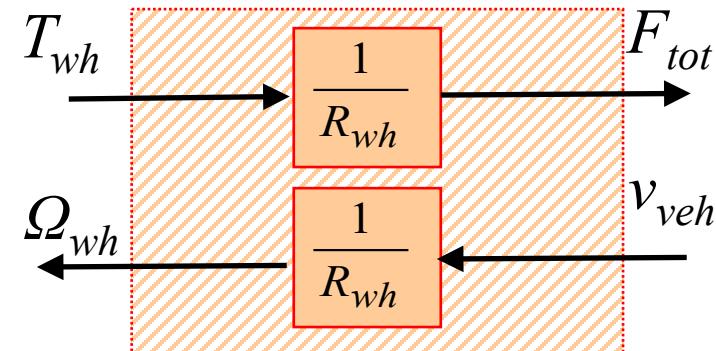
- Direct inversion vs. Undirect inversion -

- Undirect inversion: controller for the inversion of accumulation elements



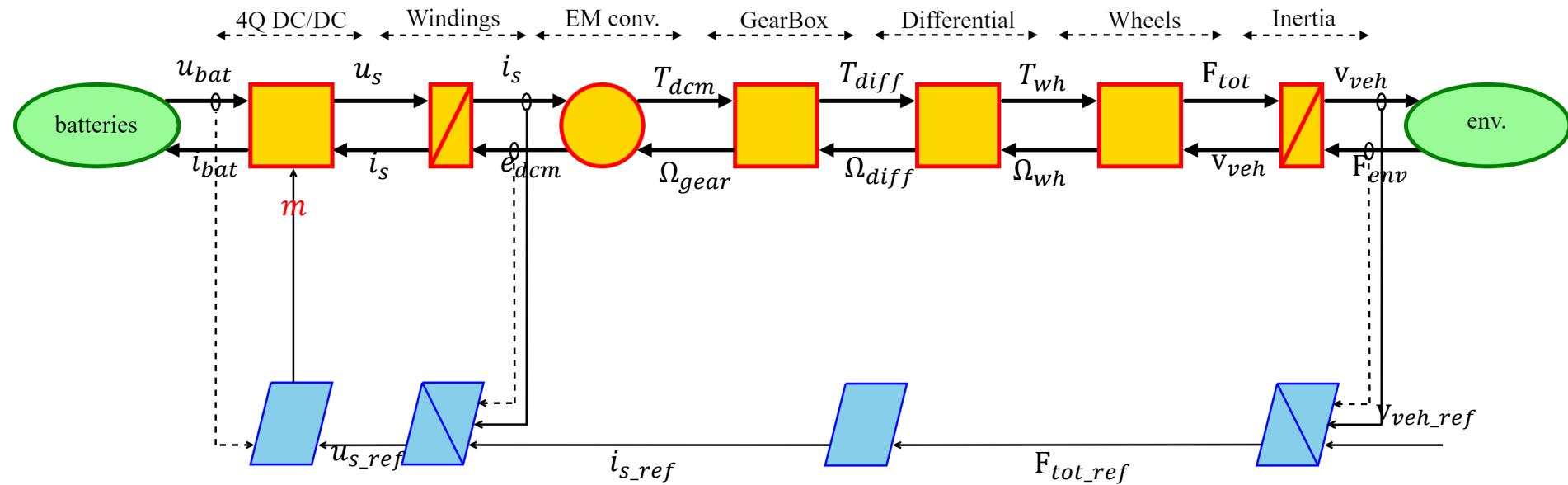
- Direct inversion vs. Undirect inversion -

- Direct inversion: pure model inversion



- To the practical implementation -

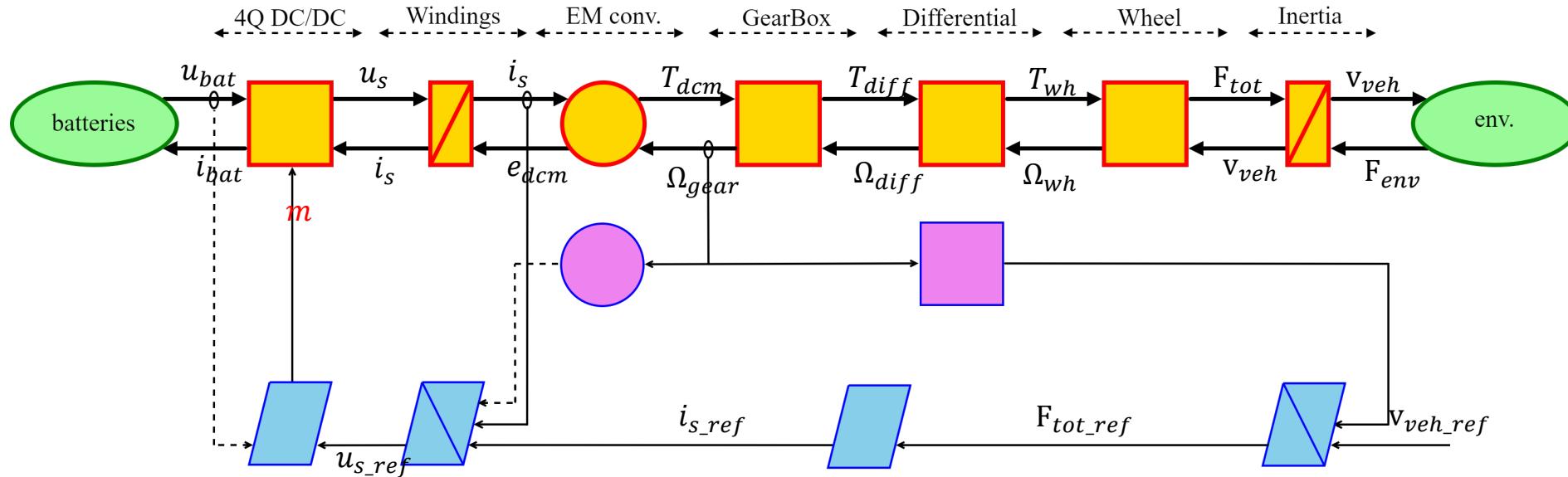
- Merge of all «Direct Inversion» elements that are series connected



$$F_{tot} = \underbrace{K_{dcm} \cdot K_{gear} \cdot K_{diff} \cdot K_{wh}}_K \cdot i_s \Rightarrow i_{s_ref} = \frac{1}{\underbrace{K_{dcm} \cdot K_{gear} \cdot K_{diff} \cdot K_{wh}}_{1/K}} \cdot F_{tot_ref}$$

- To the practical implementation -

- Check if all measures can be implemented
- For measures that are not physically accessible, check if models behind EMR symbol can be used for estimation,

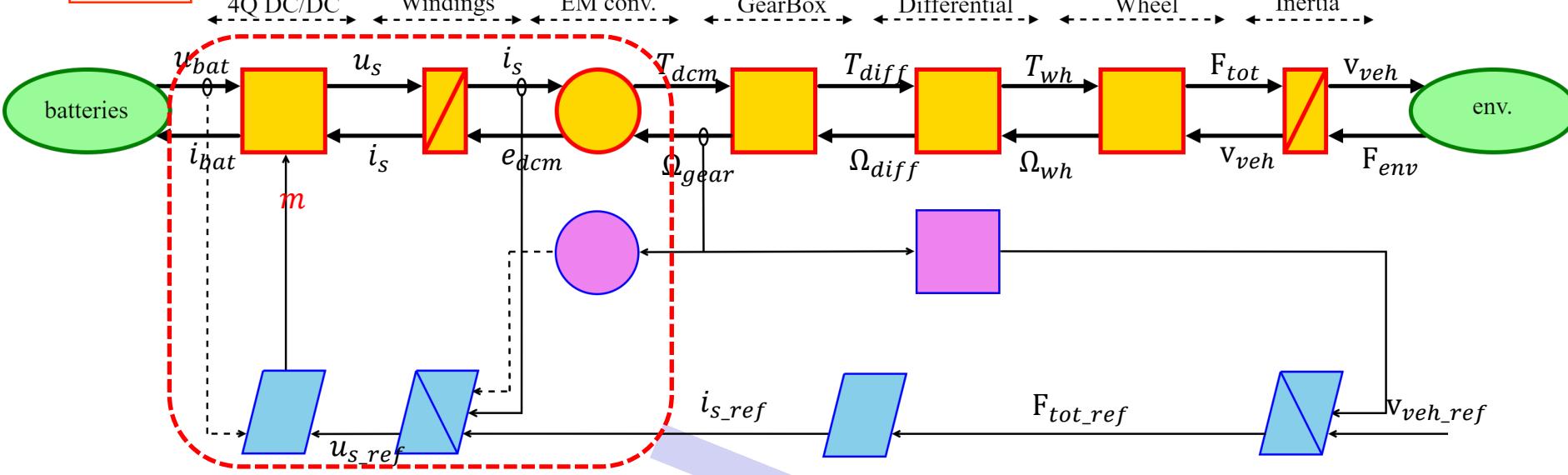


EMR and control of an EV

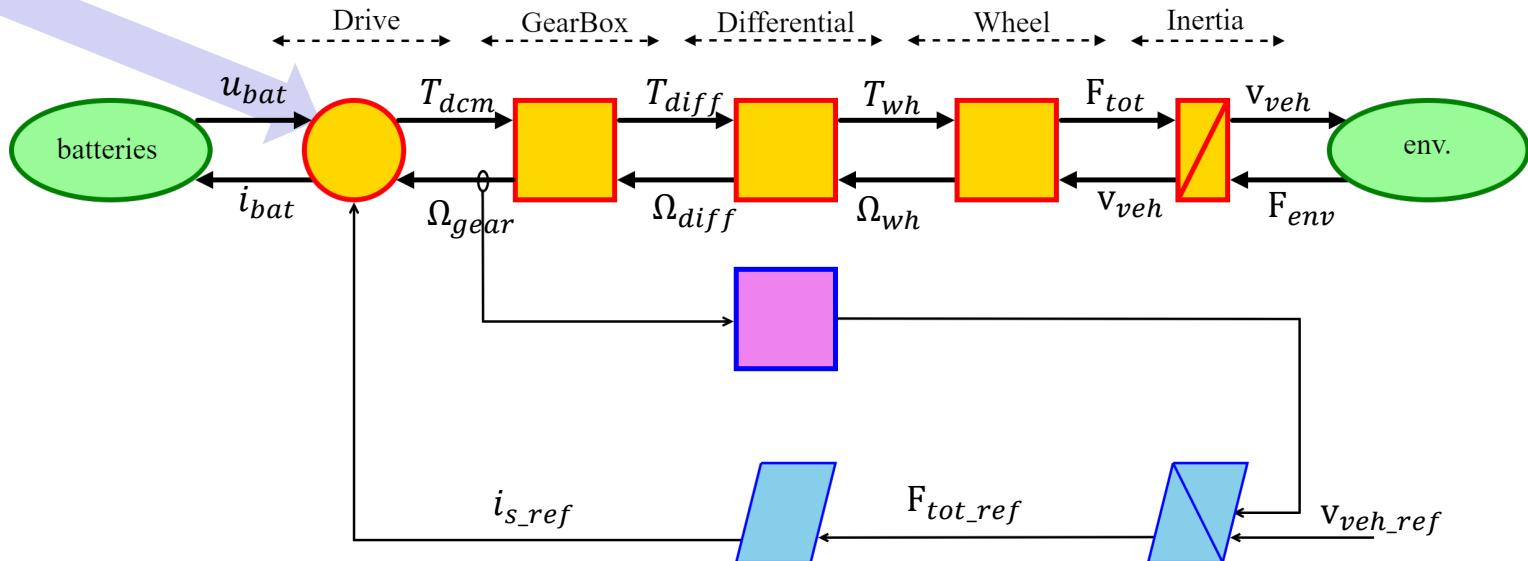
- Model simplification -

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model reduction



- For large simulation time, in case model for the converter and the motor windings require too small time steps.

Conclusion

- ❖ EMR: powerful approach for describing and representing complex and multiphysic systems
- ❖ IBC: based on inversion rules, allow a safe design for control design
- ❖ Case considered in this contribution is based on DC motors, Anyway, such an approach is obviously applicable with benefits to more modern technologies:
 - Induction motor(s)
 - PM synchronous motor(s)
 - etc.
- ❖ EMR for EVs:
 - In the 1st step: Simplified model using DC motor with chopper
(for IM, PMSM: the same principles are applied)
 - EMR: identification of functions linked to sub-systems,
 - Inversion-based control, from inversion of EMR functions
 - First step for Simulation implementation: in Matlab/Simulink using EMR library (for example)

- Authors -



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- References -

[Bouscayrol 2012] A. Bouscayrol, J. P. Hautier, B. Lemaire-Semail, "Graphic Formalisms for the Control of Multi-Physical Energetic Systems", Systemic Design Methodologies for Electrical Energy, tome 1, Analysis, Synthesis and Management, Chapter 3, ISTE Willey editions, October 2012, ISBN: 9781848213883

[Bouscayrol 2015] A. Bouscayrol, Philippe Delarue, Walter Lhomme, Betty Lemaire-Semail, "Teaching drive control using Energetic Macroscopic Representation – From maximal to practical control schemes", EPE'15 ECCE Europe, Geneva (Switzerland), September 2015.

[Bouscayrol 2023] A. Bouscayrol, B. Lemaire-Semail, "Energetic Macroscopic Representation and Inversion-Based Control ", Encyclopedia of electrical and electronic power engineering, Vol. 3, pp 365-375, Elsevier, DOI : 10.1016/B978-0-12-821204-2.00117-3, ISBN : 978-0-12-823211-8, 2023

[Nguyen 2020] C. T. P. Nguyen, J. P. F. Trovao, B. -H. Nguyen and M. C. Ta, "Powertrain Analysis of an All-Wheel-Drive Off-Road Electric Vehicle," 2019 IEEE Vehicle Power and Propulsion Conference (VPPC), Hanoi, Vietnam, 2019, pp. 1-6, doi: 10.1109/VPPC46532.2019.8952550.

[Nguyen 2019] B. -H. Nguyen, R. German, J. P. F. Trovão and A. Bouscayrol, "Real-Time Energy Management of Battery/Supercapacitor Electric Vehicles Based on an Adaptation of Pontryagin's Minimum Principle," in IEEE Transactions on Vehicular Technology, vol. 68, no. 1, pp. 203-212, Jan. 2019, doi: 10.1109/TVT.2018.2881057.